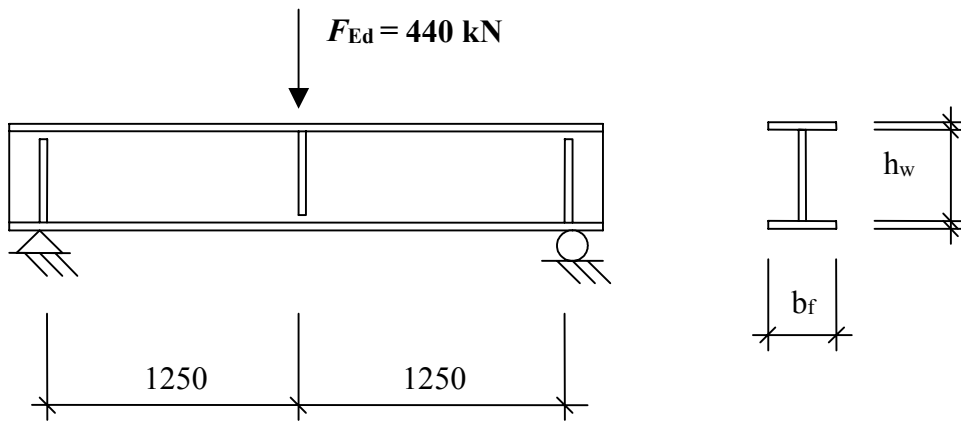


CALCULATION SHEET

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DESIGN EXAMPLE 7 – SHEAR RESISTANCE OF PLATE GIRDER

Design a plate girder with respect to shear resistance. The girder is a simply supported I-section with a span according to the figure below. The top flange is laterally restrained.



Use material grade 1.4462, hot rolled.

$$f_y = 460 \text{ N/mm}^2$$

$$E = 200\,000 \text{ N/mm}^2$$

Try a cross section with

- Flanges: $12 \times 200 \text{ mm}^2$
- Web: $4 \times 500 \text{ mm}^2$
- Stiffeners: $12 \times 98 \text{ mm}^2$
- Weld throat thickness: 4 mm

Structural analysis

Maximum shear and bending moment are obtained as

$$V_{Ed} = \frac{F_{Ed}}{2} = \frac{440}{2} = 220 \text{ kN}$$

$$M_{Ed} = \frac{F_{Ed}L}{4} = \frac{440 \times 2,5}{4} = 275 \text{ kNm}$$

Partial safety factors

$$\gamma_{M0} = 1,1$$

$$\gamma_{M1} = 1,1$$

Classification of the cross-section


$$\varepsilon = \sqrt{\frac{235\,200}{460\,210}} = 0,698$$

Table 3.1
Section 3.2.4

Table 2.1

Section 4.3

Table 4.2

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Web, subject to bending

$$\frac{c}{t\varepsilon} = \frac{500 - 2 \times \sqrt{2} \times 4}{4 \times 0,698} = 175 > 74,8, \text{ therefore the web is Class 4.}$$

Table 4.2

Flange, subject to compression

$$\frac{c}{t\varepsilon} = \frac{200 - 4 - 2 \times \sqrt{2} \times 4}{2 \times 12 \times 0,698} = 11,0 \leq 11,0, \text{ therefore the compression flange is Class 3.}$$

Table 4.2

Thus, overall classification of cross-section is Class 4.

Shear resistance

Section 5.4.3

The shear buckling resistance requires checking when $h_w/t_w \geq \frac{23}{\eta} \varepsilon \sqrt{k_\tau}$ for vertically stiffened webs.

$$a/h_w = 1250/500 > 1, \text{ hence}$$

$$k_\tau = 5,34 + 4 \left(\frac{h_w}{a} \right)^2 = 5,34 + 4 \left(\frac{500}{1250} \right)^2 = 5,98$$

Eq. 5.16a

EN 1993-1-4 recommended value for $\eta = 1,2$

Section 5.4.3

$$h_w/t_w = \frac{500}{4} = 125 \geq \frac{23}{1,2} 0,698 \sqrt{5,98} = 32,7$$

Therefore the shear buckling resistance has to be checked. It is obtained as

$$V_{b,Rd} = V_{bw,Rd} + V_{bf,Rd} \leq \frac{\eta f_{yw} h_w t_w}{\sqrt{3} \gamma_{M1}} = \frac{1,2 \times 460 \times 500 \times 4}{\sqrt{3} \times 1,1} = 579,47 \text{ kN}$$

Eq. 5.12a

$$V_{bw,Rd} = \frac{\chi_w f_{yw} h_w t_w}{\sqrt{3} \gamma_{M1}}$$

Eq. 5.12b

$$\chi_w = \eta = 1,2 \quad \text{for } \bar{\lambda}_w \leq 0,60/\eta = 0,5$$

Eq. 5.13a

$$\chi_w = 0,11 + \frac{0,64}{\bar{\lambda}_w} - \frac{0,05}{\bar{\lambda}_w^2} \quad \text{for } \bar{\lambda}_w > 0,60/\eta = 0,5$$

Eq. 5.13b


$$\bar{\lambda}_w = \left(\frac{h_w}{37,4 t_w \varepsilon \sqrt{k_\tau}} \right)$$

Eq. 5.15

$$\bar{\lambda}_w = \left(\frac{500}{37,4 \times 4 \times 0,698 \times \sqrt{5,98}} \right) = 1,958 > 0,60/\eta = 0,5$$

Hence the contribution from the web is obtained as:

$$\chi_w = 0,11 + \frac{0,64}{1,958} - \frac{0,05}{1,958^2} = 0,424$$

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$$V_{bw,Rd} = \frac{\chi_w f_{yw} h_w t_w}{\sqrt{3} \gamma_{M1}} = \frac{0,424 \times 460 \times 500 \times 4}{\sqrt{3} \times 1,1} = 204,74 \text{ kN}$$

The contribution from the flanges may be utilised if the flanges are not fully utilised in withstanding the bending moment. The bending resistance of a cross section consisting of the flanges only is obtained as

$$M_{f,Rd} = 12 \times 200 \times \frac{460}{1,1} \times (500 + 12) = 513,86 \text{ kNm}$$

$M_{f,Rd} > M_{Ed} = 275 \text{ kNm}$, therefore the flanges can contribute to the shear buckling resistance.

$$V_{bf,Rd} = \frac{b_f t_f^2 f_{yf}}{c \gamma_{M1}} \left[1 - \left[\frac{M_{Ed}}{M_{f,Rd}} \right]^2 \right]$$

$$c = a \left[0,17 + \frac{3,5 b_f t_f^2 f_{yf}}{t_w h_w^2 f_{yw}} \right] \text{ but } \frac{c}{a} \leq 0,65$$

$$= 1250 \times \left[0,17 + \frac{3,5 \times 200 \times 12^2 \times 460}{4 \times 500^2 \times 460} \right] = 338 \text{ mm} < 0,65 \times 1250 = 812 \text{ mm}$$

$$V_{bf,Rd} = \frac{200 \times 12^2 \times 460}{338 \times 1,1} \times \left[1 - \left[\frac{275}{513,86} \right]^2 \right] = 25,43 \text{ kN}$$

$$V_{b,Rd} = V_{bw,Rd} + V_{bf,Rd} = 230,17 \text{ kN} \leq 579,47 \text{ kN}$$

Transverse stiffeners

The transverse stiffeners have to be checked for crushing and flexural buckling using $\alpha = 0,49$, $\bar{\lambda}_0 = 0,2$. An effective cross section consisting of the stiffeners and parts of the web is then used. The part of the web included is $11 \varepsilon t_w$ wide, therefore the cross section of the transverse stiffener is Class 3.

$a/h_w = 1250/500 = 2,5 \geq \sqrt{2}$, hence the second moment of area of the intermediate stiffener has to fulfil

$$I_{st} \geq 0,75 h_w t_w^3 = 0,75 \times 500 \times 4^3 = 24000 \text{ mm}^4$$

$$I_{st} = 2 \times \frac{(11 \times 0,698 \times 4) \times 4^3}{12} + \frac{12 \times 200^3}{12} = 8,00 \times 10^6 \text{ mm}^4, \text{ hence fulfilled.}$$

The crushing resistance is obtained as

$$N_{c,Rd} = A_s f_y / \gamma_{M0}$$

$$A_s = (12 \times 200 + 11 \times 0,698 \times 4 \times 2) = 2461,42 \text{ mm}^2$$

Section 5.4.3


Eq. 5.17

Section 5.4.5

Eq. 5.37

Eq. 5.37

Eq. 4.25

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$$N_{c,Rd} = 2461,42 \times 460 / 1,1 = 1029,32 \text{ kN}$$

The flexural buckling resistance is obtained as

$$N_{b,Rd} = \chi A_s f_y / \gamma_{M1}$$

$$\chi = \frac{1}{\varphi + [\varphi^2 - \bar{\lambda}^2]^{0,5}} \leq 1$$

$$\varphi = 0,5 \left(1 + \alpha(\bar{\lambda} - \bar{\lambda}_0) + \bar{\lambda}^2 \right)$$

$$\bar{\lambda} = \frac{L_{cr}}{i} \frac{1}{\pi} \sqrt{\frac{f_{yw}}{E}}$$

$$L_{cr} = 0,75h_w = 0,75 \times 500 = 375 \text{ mm}$$

$$\bar{\lambda} = \frac{375}{\sqrt{\frac{8 \times 10^6}{2461,42}}} \frac{1}{\pi} \sqrt{\frac{460 \times 1}{200000}} = 0,100$$

$$\varphi = 0,5 \times \left(1 + 0,49 \times (0,100 - 0,2) + 0,100^2 \right) = 0,481$$

$$\chi = \frac{1}{0,481 + [0,481^2 - 0,100^2]^{0,5}} = 1,05 > 1 \Rightarrow \chi = 1,0$$

Since $N_{b,Rd} = N_{c,Rd} > N_{Ed}$, the transverse stiffeners are sufficient.

Interaction shear and bending

If the utilization of shear resistance, expressed as the factor $\bar{\eta}_3$, exceeds 0,5, the combined effect of bending and shear has to be checked.

$$\bar{\eta}_3 = \frac{V_{Ed}}{V_{bw,Rd}} \leq 1,0$$

$$\bar{\eta}_3 = \frac{220}{204,74} = 1,075 > 0,5, \text{ therefore interaction has to be considered.}$$

The condition is

$$\bar{\eta}_1 + \left(1 - \frac{M_{f,Rd}}{M_{pl,Rd}} \right) (2\bar{\eta}_3 - 1)^2 \leq 1,0 \text{ for } \bar{\eta}_1 \geq \frac{M_{f,Rd}}{M_{pl,Rd}}$$

Where:

$$\bar{\eta}_1 = \frac{M_{Ed}}{M_{pl,Rd}}$$

Eq. 5.2a

Eq. 5.3

Eq. 5.4

Eq. 5.5a


Section 5.4.5

Section 5.4.3

Eq. 5.23

Eq. 5.21

Eq. 5.22

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Calculation of effective cross-section properties.

The flanges are Class 3 and hence fully effective.

The depth of the web has to be reduced with the reduction factor ρ , welded web.

$$\rho = \frac{0,772}{\bar{\lambda}_p} - \frac{0,125}{\bar{\lambda}_p^2} \leq 1$$

Eq. 4.1a

$$\bar{\lambda}_p = \frac{\bar{b}/t}{28,4\epsilon\sqrt{k_\sigma}} \quad \text{where } \bar{b} = d = 500 - 2 \times 4 \times \sqrt{2} = 488,68 \text{ mm}$$

Eq. 4.2

Assuming linearly varying, symmetric stress distribution within the web,

$$\psi = \frac{\sigma_2}{\sigma_1} = -1$$

$$\Rightarrow k_\sigma = 23,9$$

Table 4.3

$$\bar{\lambda}_p = \frac{488,68/4}{28,4 \times 0,698 \times \sqrt{23,9}} = 1,26$$

$$\rho = \frac{0,772}{1,26} - \frac{0,125}{1,26^2} = 0,534 \leq 1$$

$$b_{\text{eff}} = \rho b_c = \rho \bar{b} / (1 - \psi) = 0,534 \times 488,68 / (1 - (-1)) = 130,48$$

Table 4.3

$$b_{e1} = 0,4b_{\text{eff}} = 0,4 \times 130,48 = 52,19 \text{ mm}$$

Table 4.3

$$b_{e2} = 0,6b_{\text{eff}} = 0,6 \times 130,48 = 78,29 \text{ mm}$$


Calculate effective section modulus under bending.

e_i is taken as positive from the centroid of the upper flange and downwards.

$$A_{\text{eff}} = \sum_i A_i = b_f t_f \times 2 + b_{e1} t_w + b_{e2} t_w + (h_w / 2) t_w = 6321,92 \text{ mm}^2$$

$$e_{\text{eff}} = \frac{1}{A_{\text{eff}}} \sum_i A_i e_i = \frac{1}{A_{\text{eff}}} [b_f t_f (0) + b_f t_f (h_w + t_f)] + [b_{e1} t_w (0,5(b_{e1} + t_f)) + b_{e2} t_w (0,5(h_w + t_f) - b_{e2} / 2) + (h_w / 2) t_w (0,75 h_w + 0,5 t_f)] = 266,44 \text{ mm}$$

$$I_{\text{eff}} = \sum_i I_i + \sum_i A_i (e_{\text{eff}} - e_i)^2 = 2 \times \frac{b_f t_f^3}{12} + \frac{t_w b_{e1}^3}{12} + \frac{t_w b_{e2}^3}{12} + \frac{t_w (h_w / 2)^3}{12} + b_f t_f (e_{\text{eff}} - 0)^2 + b_f t_f [e_{\text{eff}} - (h_w + t_f)]^2 + b_{e1} t_w [e_{\text{eff}} - 0,5(b_{e1} + t_f)]^2 + b_{e2} t_w [e_{\text{eff}} - 0,5(h_w + t_f + b_{e2})]^2 + (h_w / 2) t_w [e_{\text{eff}} - (0,75 h_w + 0,5 t_f)]^2 = 3,459 \times 10^8 \text{ mm}^4$$

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$$\bar{\eta}_1 = \frac{M_{Ed}}{M_{pl,Rd}}$$

$$M_{f,Rd} = 513,86 \text{ kNm (Sheet 3)}$$

$M_{pl,Rd}$ is the plastic resistance of the cross-section.

$$M_{pl,Rd} = M_{f,Rd} + \frac{t_w h_w^2 f_y}{4 \gamma_{M0}} = 513,86 + \frac{4 \times 500^2 \times 460}{4 \times 1,1 \times 10^6} = 618,40 \text{ kNm}$$

Evaluate conditions

$$M_{Ed} = 275 \text{ kNm, hence:}$$

$$\bar{\eta}_1 = \frac{275}{618,40} = 0,44 \leq 1,0 \text{ OK}$$

$\bar{\eta}_1$ fulfils its condition. Now it remains to check the interaction.

$$\bar{\eta}_1 + \left(1 - \frac{M_{f,Rd}}{M_{pl,Rd}}\right) \left(2\bar{\eta}_3 - 1\right)^2 = 0,44 + \left(1 - \frac{513,86}{618,40}\right) \left((2 \times 1,075) - 1\right)^2 = 0,664 < 1,0$$

It therefore follows that under the conditions given, the resistance of the plate girder is sufficient with respect to shear, bending as well as interaction between shear and bending.

Eq. 5.22